

## DRILLSHIP GLOBAL STRENGTH ANALYSIS UNDER OBLIQUE EQUIVALENT DESIGN WAVES

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### ABSTRACT

*For the design of off-shore drillships, one of the criteria for initial strength assessment is based on global strength analysis under oblique equivalent design waves (EDW). The global strength procedure is based on an iterative approach, taking into account the ship hull external shape non-linearities, for a set of heading angles and design wave heights according to shipbuilding classification societies rules. This study analyzes the global strength of a drillship having 210 m length, for two relevant displacement cases. The numerical analysis is done by own code, delivering the ship-wave relative position and the generated wave loads. The results are compared to the limit values from several shipbuilding rules and the initial drillship strength is evaluated.*

**Keywords:** drillship, equivalent design wave, oblique waves, initial strength assessment.

### 1. INTRODUCTION

For the initial design of an off-shore drillship several evaluation criteria are to be checked according to the shipbuilding rules [9].

In previous studies we have presented the hydroelastic response analysis in irregular head waves [6] and the seakeeping response analysis in irregular oblique waves [8] of a large drillship, with maximum length of 210 m, several operation speeds and sea states, and two different loading conditions.

For the same drillship, this analysis is focused on the strength evaluation in the initial design stage under the loads of the equivalent design waves (EDW) [10],[15], according to the shipbuilding classification societies rules [9].

For the strength analysis we use an own developed iterative procedure, modelling the external hull shape non-linearities, for different oblique EDW waves, by program P\_QSW. The theoretical elements are in detail presented in reference [7]. The results are delivering the EDW wave limits from global strength criteria.

### 2. THE DRILLSHIP DATA

The drillship main data are:

- the drillship characteristics and the two loading cases (Table 1) [6],[8];
- the external shape lines of the drillship (Fig.1) [6];
- the mass distributions are considered from reference [6];

**Table 1.** The drillship characteristics and the two loading cases [6],[8]

$L$ [m]	210	$\Delta_2$ [t]	67868.5
$B$ [m]	34	$T_{m2}$ [m]	12.085
$H$ [m]	17.5	$x_{G2}$ [m]	99.85
$\rho$ [t/m <sup>3</sup> ]	1.025	$y_{G2}$ [m]	0
$g$ [m/s <sup>2</sup> ]	9.81	$z_{G2}$ [m]	10.12
$N_e$	40	$z_{R2}$ [m]	10.10
$\Delta_1$ [t]	59217.5	$\delta x$ [m]	0.7 ÷ 14.7
$T_{m1}$ [m]	10.684	$\mu$ [deg]	0 ÷ 75
$x_{G1}$ [m]	100.56	$\delta\mu$ [deg]	15
$y_{G1}$ [m]	0	$h_w$ [m]	0 ÷ 9.896(10)
$z_{G1}$ [m]	10.08	EDW	$\lambda_1=L$
$z_{R1}$ [m]	10.10	length	$\lambda=\lambda_1\cos\mu$

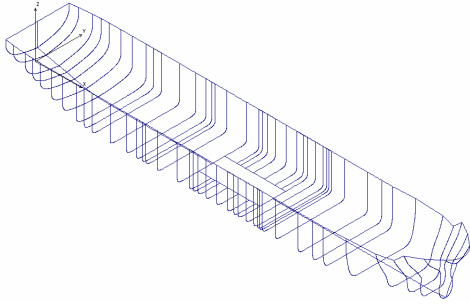


Fig.1. The external shape lines of the drillship

**3. THE STRENGTH ANALYSIS FOR THE FIRST LOADING CASE**

Table 2. First case Δ<sub>1</sub> equilibrium parameters

drillship l	u[deg]	sagging			hogging		
		h <sub>w</sub> [m]	0	5	9.896	0	5
0	x <sub>F</sub> [m]	96.419	96.850	97.237	96.419	95.967	95.578
	y <sub>F</sub> [m]	0.000	0.000	0.000	0.000	0.000	0.000
	T <sub>m</sub> [m]	10.685	10.751	10.778	10.685	10.584	10.449
	θ[rad]	-0.002	0.004	0.010	-0.002	-0.009	-0.016
	φ[rad]	0.000	0.000	0.000	0.000	0.000	0.000
	z <sub>H</sub> [m]	3.697	3.923	4.471	3.697	3.847	4.283
	z <sub>w</sub> [m]	3.697	3.923	4.471	3.697	3.847	4.283
15	x <sub>F</sub> [m]	96.419	96.845	97.227	96.419	95.971	95.584
	y <sub>F</sub> [m]	0.000	0.001	0.001	0.000	-0.001	-0.002
	T <sub>m</sub> [m]	10.685	10.751	10.778	10.685	10.584	10.451
	θ[rad]	-0.002	0.004	0.010	-0.002	-0.009	-0.016
	φ[rad]	0.000	0.000	0.000	0.000	-0.001	-0.002
	z <sub>H</sub> [m]	3.697	3.924	4.475	3.697	3.849	4.288
	z <sub>w</sub> [m]	3.697	3.924	4.475	3.697	3.849	4.288
30	x <sub>F</sub> [m]	96.419	96.845	97.227	96.419	95.986	95.607
	y <sub>F</sub> [m]	0.000	0.001	0.001	0.000	-0.003	-0.005
	T <sub>m</sub> [m]	10.685	10.751	10.778	10.685	10.586	10.457
	θ[rad]	-0.002	0.004	0.010	-0.002	-0.009	-0.015
	φ[rad]	0.000	0.000	0.000	0.000	-0.002	-0.004
	z <sub>H</sub> [m]	3.697	3.924	4.475	3.697	3.854	4.306
	z <sub>w</sub> [m]	3.697	3.924	4.475	3.697	3.854	4.306
45	x <sub>F</sub> [m]	96.419	96.781	97.098	96.419	96.025	95.664
	y <sub>F</sub> [m]	0.000	0.003	0.003	0.000	-0.004	-0.008
	T <sub>m</sub> [m]	10.685	10.751	10.785	10.685	10.590	10.472
	θ[rad]	-0.002	0.003	0.008	-0.002	-0.008	-0.014
	φ[rad]	0.000	0.002	0.002	0.000	-0.003	-0.007
	z <sub>H</sub> [m]	3.697	3.938	4.521	3.697	3.866	4.349
	z <sub>w</sub> [m]	3.697	3.938	4.521	3.697	3.866	4.349
60	x <sub>F</sub> [m]	96.419	96.656	96.844	96.419	96.135	95.832
	y <sub>F</sub> [m]	0.000	0.005	0.007	0.000	-0.006	-0.013
	T <sub>m</sub> [m]	10.685	10.750	10.792	10.685	10.602	10.513
	θ[rad]	-0.002	0.001	0.004	-0.002	-0.006	-0.011
	φ[rad]	0.000	0.003	0.005	0.000	-0.005	-0.011
	z <sub>H</sub> [m]	3.697	3.958	4.581	3.697	3.893	4.443
	z <sub>w</sub> [m]	3.697	3.958	4.581	3.697	3.893	4.443
75	x <sub>F</sub> [m]	96.419	96.180	95.943	96.419	96.641	95.973
	y <sub>F</sub> [m]	0.000	0.006	0.012	0.000	-0.005	-0.015
	T <sub>m</sub> [m]	10.685	10.727	10.756	10.685	10.636	10.539
	θ[rad]	-0.002	-0.006	-0.010	-0.002	0.001	-0.009
	φ[rad]	0.000	0.005	0.011	0.000	-0.004	-0.012
	z <sub>H</sub> [m]	3.697	3.921	4.446	3.697	3.889	4.489
	z <sub>w</sub> [m]	3.697	3.921	4.446	3.697	3.889	4.489

In the case of first loading case of the drillship, using the iterative procedure from reference [7], the next results are obtained:

- Table 2, the equilibrium parameters drillship-EDW wave: *h<sub>w</sub>* wave height, *μ* heading angle, *x<sub>F</sub>*, *z<sub>F</sub>* the wave medium plane centre, *d<sub>m</sub>*, *θ*, *φ* the sinkage, trim and roll angles of the wave medium plane into the reference of the drillship base plan, *z<sub>H</sub>* the pressure centre vertical position, *ζ<sub>w</sub>* wave free surface in the (*x,y,z*) drillship coordinates system (1);

$$\zeta_w(x,y) = T_m + (x - x_F) \cdot \theta + (y - y_F) \cdot tg(\phi) \pm \frac{h_w}{2} \cos\left[\frac{2\pi}{\lambda}(x \cos \mu + y \sin \mu)\right] \quad (1)$$

$$h_{w \max} = 10.75 - \left(\frac{300 - L}{100}\right)^{1.5}$$

- for hogging and sagging, heading angle *μ* = 0, 45, 75 deg., the following sectional efforts are obtained: *M<sub>v</sub>*[kNm] vertical bending moment (Figs.2.a,b.1-3); *T<sub>v</sub>*[kN] vertical shear force (Figs.3.a,b.1-3); *M<sub>h</sub>* [kNm] horizontal bending moment (Figs.4.a,b.1-3); *T<sub>h</sub>*[kN] horizontal shear force (Figs.5.a,b.1-3); *M<sub>t</sub>* [kNm] torsional moment (Figs.6.a,b.1-3); - Table 3, the maximum wave EWD induced loads, for reference *h<sub>w max</sub>* = 9.896 m, *μ*=0, 15, 30, 45, 60, 75 deg., sagging and hogging conditions, first case Δ<sub>1</sub>.

Table 3. First case Δ<sub>1</sub> maximum wave EDW induced loads

sagging						
μ	0	15	30	45	60	75
λ/L	1	0.966	0.866	0.707	0.500	0.259
M <sub>v</sub>	3.73E+6	3.71E+6	3.65E+6	3.50E+6	3.06E+6	1.27E+6
T <sub>v</sub>	8.01E+4	7.98E+4	7.87E+4	7.60E+4	6.85E+4	3.84E+4
M <sub>h</sub>	0	7.07E+4	1.51E+5	2.55E+5	4.12E+5	5.23E+5
T <sub>h</sub>	0	1.97E+3	4.19E+3	7.02E+3	1.11E+4	1.81E+4
M <sub>t</sub>	0	1.06E+5	2.26E+5	3.77E+5	5.88E+5	6.84E+5

hogging						
μ	0	15	30	45	60	75
λ/L	1	0.966	0.866	0.707	0.500	0.259
M <sub>v</sub>	1.53E+6	1.52E+6	1.47E+6	1.36E+6	1.04E+6	7.98E+5
T <sub>v</sub>	3.99E+4	3.97E+4	3.89E+4	3.70E+4	3.12E+4	2.69E+4
M <sub>h</sub>	0	9.20E+4	1.95E+5	3.25E+5	4.97E+5	5.56E+5
T <sub>h</sub>	0	2.99E+3	6.34E+3	1.06E+4	1.62E+4	1.82E+4
M <sub>t</sub>	0	8.92E+4	1.91E+5	3.24E+5	5.27E+5	6.18E+5

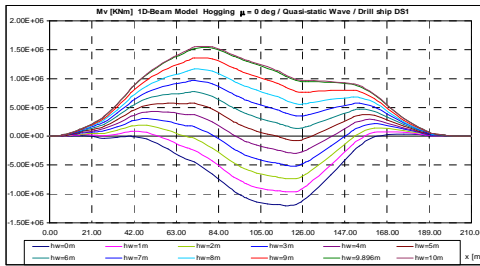


Fig.2.a.1 Case  $\Delta_1, M_v$  [kNm],  $\mu=0$  deg., hogg.

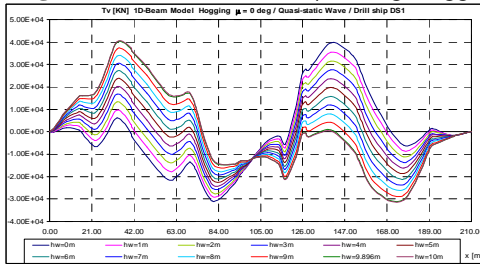


Fig.3.a.1 Case  $\Delta_1, T_v$  [kN],  $\mu=0$  deg., hogg.

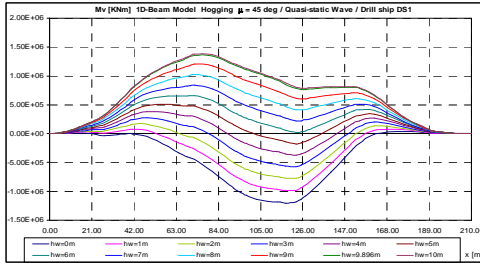


Fig.2.a.2 Case  $\Delta_1, M_v$  [kNm],  $\mu=45$  deg., hogg.

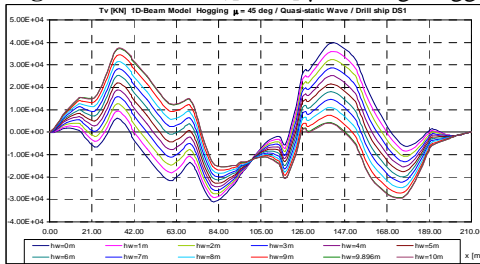


Fig.3.a.2 Case  $\Delta_1, T_v$  [kN],  $\mu=45$  deg., hogg.

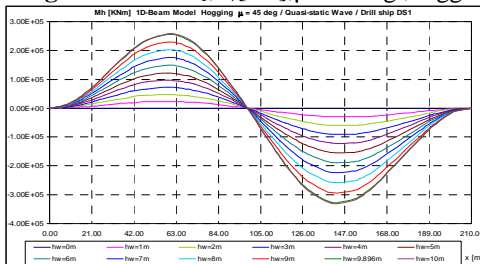


Fig.4.a.2 Case  $\Delta_1, M_h$  [kNm],  $\mu=45$  deg., hogg.

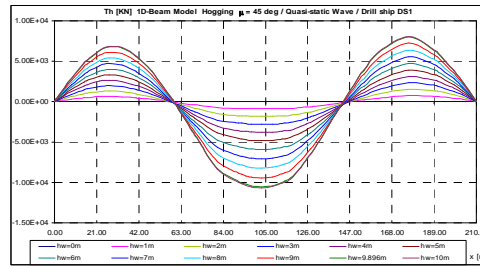


Fig.5.a.2 Case  $\Delta_1, T_h$  [kN],  $\mu=45$  deg., hogg.

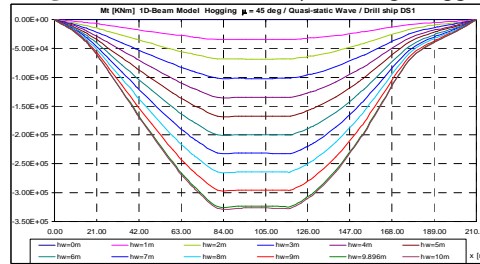


Fig.6.a.2 Case  $\Delta_1, M_h$  [kNm],  $\mu=45$  deg., hogg.

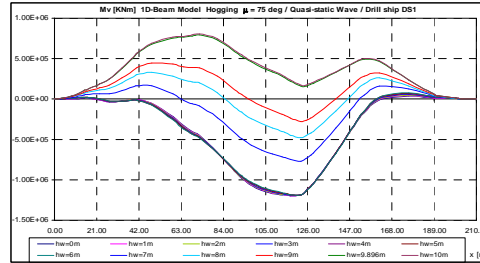


Fig.2.a.3 Case  $\Delta_1, M_v$  [kNm],  $\mu=75$  deg., hogg.

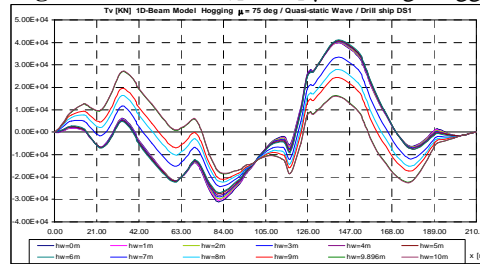


Fig.3.a.3 Case  $\Delta_1, T_v$  [kN],  $\mu=75$  deg., hogg.

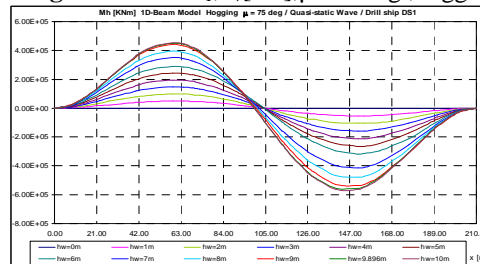


Fig.4.a.3 Case  $\Delta_1, M_h$  [kNm],  $\mu=75$  deg., hogg.

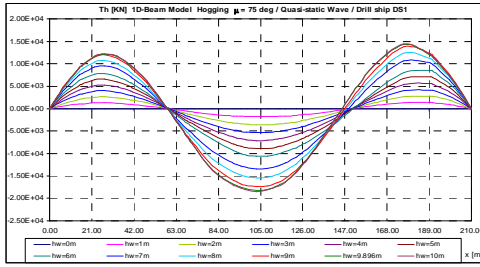


Fig.5.a.3 Case  $\Delta_1, T_h$ [kN],  $\mu=75$  deg., hogg.

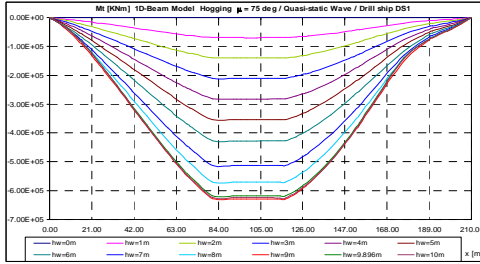


Fig.6.a.3 Case  $\Delta_1, M_h$ [kNm],  $\mu=75$  deg., hogg.

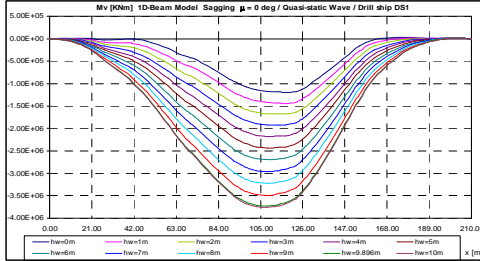


Fig.2.b.1 Case  $\Delta_1, M_v$ [kNm],  $\mu=0$  deg., sagg.

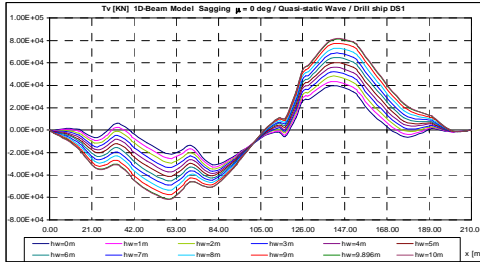


Fig.3.b.1 Case  $\Delta_1, T_v$ [kN],  $\mu=0$  deg., sagg.

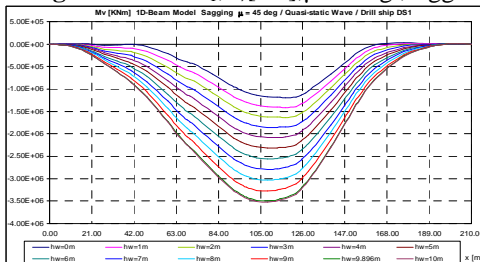


Fig.2.b.2 Case  $\Delta_1, M_v$ [kNm],  $\mu=45$  deg., sagg.

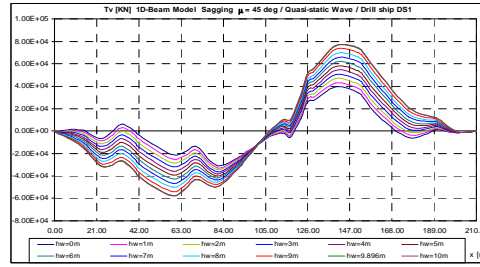


Fig.3.b.2 Case  $\Delta_1, T_v$ [kN],  $\mu=45$  deg., sagg.

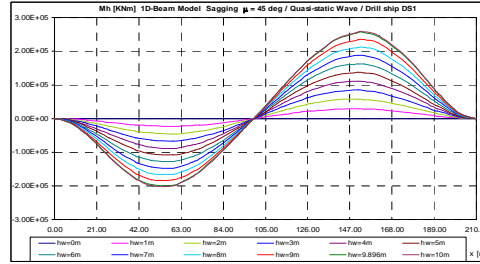


Fig.4.b.2 Case  $\Delta_1, M_h$ [kNm],  $\mu=45$  deg., sagg.

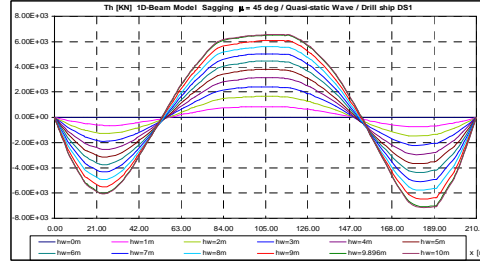


Fig.5.b.2 Case  $\Delta_1, T_h$ [kN],  $\mu=45$  deg., sagg.

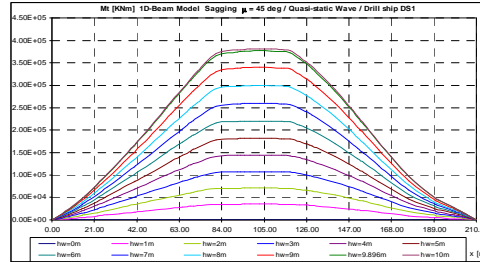


Fig.6.b.2 Case  $\Delta_1, M_h$ [kNm],  $\mu=45$  deg., sagg.

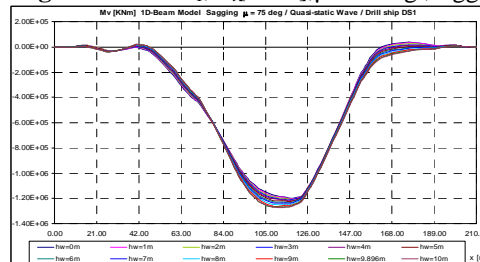


Fig.2.b.3 Case  $\Delta_1, M_v$ [kNm],  $\mu=75$  deg., sagg.

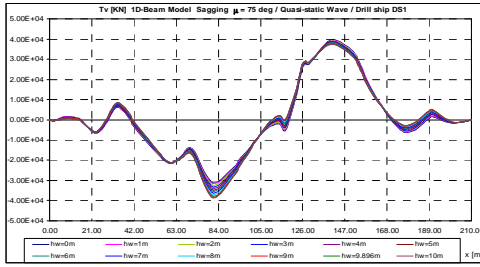


Fig.3.b.3 Case  $\Delta_1, T_v$ [kN],  $\mu=75$  deg.,sagg.

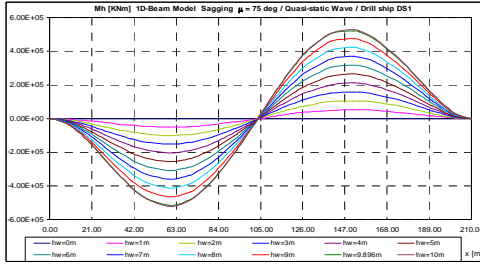


Fig.4.b.3 Case  $\Delta_1, M_v$ [kNm],  $\mu=75$  deg.,sagg.

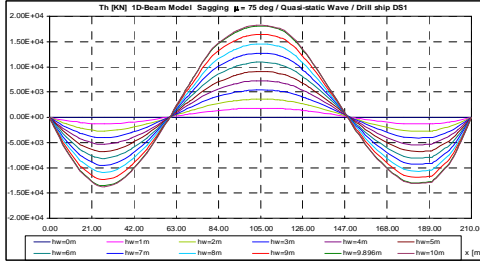


Fig.5.b.3 Case  $\Delta_1, T_h$ [kN],  $\mu=75$  deg.,sagg.

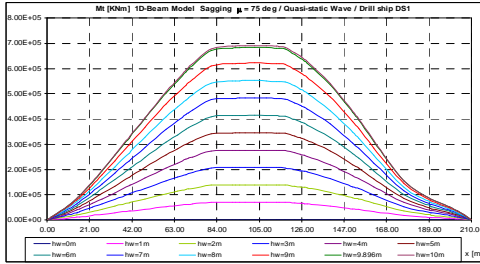


Fig.6.b.3 Case  $\Delta_1, M_h$ [kNm],  $\mu=75$  deg.,sagg.

#### 4. THE STRENGTH ANALYSIS FOR THE SECOND LOADING CASE

In the case of second loading case of the drillship, using the iterative procedure from reference [7], the next results are obtained:

- Table 4, the equilibrium parameters drillship-EDW wave (same notation as Table 2);
- Figs.7 the oblique equivalent design wave-

drillship position,  $\mu=0,15,30,45,60,75$  deg. ;  
 - for hogging and sagging, heading angle  $\mu = 0, 30, 60$  deg., the following sectional efforts are obtained:  $M_v$ [kNm] vertical bending moment (Figs.8.a,b.1-3);  $T_v$ [kN] vertical shear force (Figs.9.a,b.1-3);  $M_h$  [kNm] horizontal bending moment (Figs.10.a,b.1-3);  $T_h$ [kN] horizontal shear force (Figs.11.a,b.1-3);  $M_t$  [kNm] torsional moment (Figs.12.a,b.1-3);  
 - Table 5, the maximum wave EWD induced loads, for reference  $h_w \max = 9.896$  m,  $\mu=0, 15, 30, 45, 60, 75$  deg., sagging and hogging conditions, second case  $\Delta_2$ .

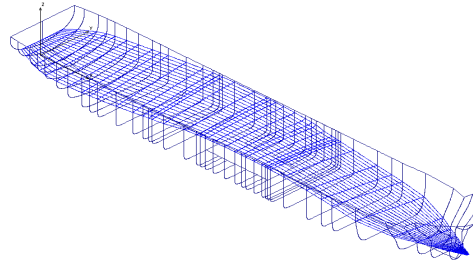
Table 4.Second case  $\Delta_2$  equilibrium parameters

drillship 2	sagging	hogging					
		0	5	9.896	0	5	9.896
$\mu$ [deg]	$h_w$ [m]	0	5	9.896	0	5	9.896
0	$x_F$ [m]	96.740	97.087	97.392	96.740	96.364	95.992
	$y_F$ [m]	0.000	0.000	0.000	0.000	0.000	0.000
	$T_m$ [m]	12.087	12.132	12.141	12.087	12.007	11.899
	$\theta$ [rad]	-0.003	0.003	0.008	-0.003	-0.010	-0.017
	$\phi$ [rad]	0.000	0.000	0.000	0.000	0.000	0.000
	$z_H$ [m]	4.182	4.377	4.862	4.182	4.318	4.719
	$z_H$ [m]	4.182	4.377	4.862	4.182	4.318	4.719
15	$x_F$ [m]	96.740	97.083	97.384	96.740	96.367	95.999
	$y_F$ [m]	0.000	0.000	0.000	0.000	-0.001	-0.002
	$T_m$ [m]	12.087	12.132	12.142	12.087	12.007	11.900
	$\theta$ [rad]	-0.003	0.003	0.008	-0.003	-0.010	-0.017
	$\phi$ [rad]	0.000	0.000	0.000	0.000	-0.001	-0.002
	$z_H$ [m]	4.182	4.378	4.866	4.182	4.320	4.723
	$z_H$ [m]	4.182	4.378	4.866	4.182	4.320	4.723
30	$x_F$ [m]	96.740	97.069	97.355	96.740	96.380	96.021
	$y_F$ [m]	0.000	0.001	0.001	0.000	-0.002	-0.004
	$T_m$ [m]	12.087	12.133	12.146	12.087	12.008	11.902
	$\theta$ [rad]	-0.003	0.003	0.008	-0.003	-0.009	-0.016
	$\phi$ [rad]	0.000	0.001	0.001	0.000	-0.002	-0.004
	$z_H$ [m]	4.182	4.382	4.879	4.182	4.324	4.740
	$z_H$ [m]	4.182	4.382	4.879	4.182	4.324	4.740
45	$x_F$ [m]	96.740	97.034	97.284	96.740	96.412	96.079
	$y_F$ [m]	0.000	0.002	0.002	0.000	-0.003	-0.007
	$T_m$ [m]	12.087	12.136	12.156	12.087	12.009	11.908
	$\theta$ [rad]	-0.003	0.002	0.006	-0.003	-0.009	-0.015
	$\phi$ [rad]	0.000	0.001	0.001	0.000	-0.003	-0.007
	$z_H$ [m]	4.182	4.390	4.908	4.182	4.335	4.776
	$z_H$ [m]	4.182	4.390	4.908	4.182	4.335	4.776
60	$x_F$ [m]	96.740	96.935	97.088	96.740	96.506	96.245
	$y_F$ [m]	0.000	0.003	0.004	0.000	-0.004	-0.009
	$T_m$ [m]	12.087	12.143	12.176	12.087	12.011	11.924
	$\theta$ [rad]	-0.003	0.000	0.003	-0.003	-0.007	-0.012
	$\phi$ [rad]	0.000	0.003	0.004	0.000	-0.004	-0.010
	$z_H$ [m]	4.182	4.407	4.963	4.182	4.358	4.858
	$z_H$ [m]	4.182	4.407	4.963	4.182	4.358	4.858
75	$x_F$ [m]	96.740	96.548	96.360	96.740	96.926	96.376
	$y_F$ [m]	0.000	0.004	0.009	0.000	-0.003	-0.011
	$T_m$ [m]	12.087	12.149	12.197	12.087	12.015	11.932
	$\theta$ [rad]	-0.003	-0.007	-0.011	-0.003	0.001	-0.009
	$\phi$ [rad]	0.000	0.004	0.009	0.000	-0.003	-0.011
	$z_H$ [m]	4.182	4.374	4.844	4.182	4.356	4.897
	$z_H$ [m]	4.182	4.374	4.844	4.182	4.356	4.897

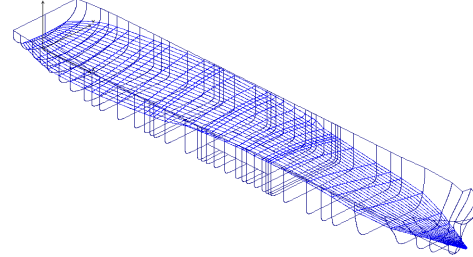
**Table 5.** Second case  $\Delta_2$  maximum wave EDW induced loads

sagging						
$\mu$	0	15	30	45	60	75
$\lambda/L$	1	0.966	0.866	0.707	0.500	0.259
$M_v$	<b>5.01E+6</b>	4.99E+6	4.92E+6	4.76E+6	4.31E+6	2.45E+6
$T_v$	<b>9.83E+4</b>	9.80E+4	9.69E+4	9.42E+4	8.66E+4	6.11E+4
$M_h$	0	8.03E+4	1.72E+5	2.91E+5	4.69E+5	<b>5.95E+5</b>
$T_h$	0	2.20E+3	4.68E+3	7.85E+3	1.27E+4	<b>2.04E+4</b>
$M_t$	0	1.06E+5	2.26E+5	3.79E+5	5.91E+5	<b>6.89E+5</b>

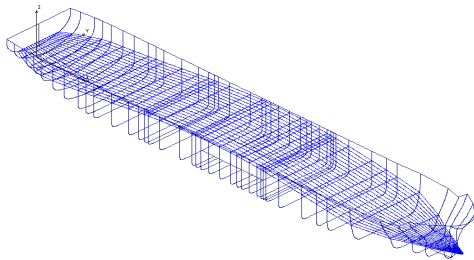
hogging						
$\mu$	0	15	30	45	60	75
$\lambda/L$	1	0.966	0.866	0.707	0.500	0.259
$M_v$	<b>1.07E+6</b>	1.06E+6	1.01E+6	9.10E+5	6.35E+5	8.90E+5
$T_v$	<b>4.26E+4</b>	4.24E+4	4.15E+4	4.08E+4	4.17E+4	4.24E+4
$M_h$	0	1.03E+5	2.19E+5	3.65E+5	5.57E+5	<b>6.24E+5</b>
$T_h$	0	3.31E+3	7.02E+3	1.17E+4	1.80E+4	<b>2.02E+4</b>
$M_t$	0	9.07E+4	1.94E+5	3.29E+5	5.34E+5	<b>6.25E+5</b>



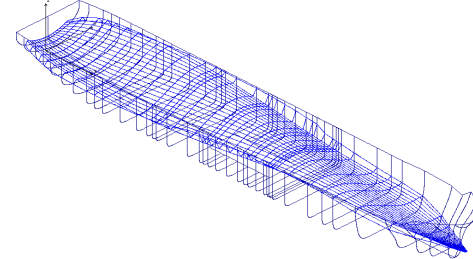
**Fig.7.a.4**  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=45deg.$ , hogging



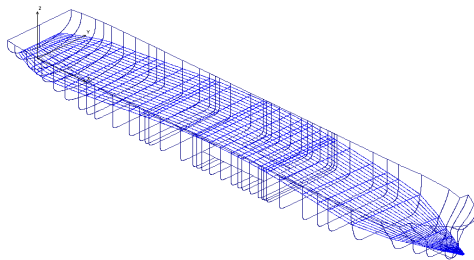
**Fig.7.a.5**  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=60deg.$ , hogging



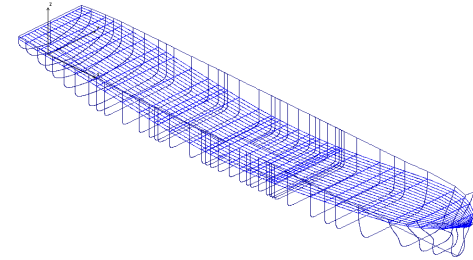
**Fig.7.a.1**  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=0 deg.$ , hogging



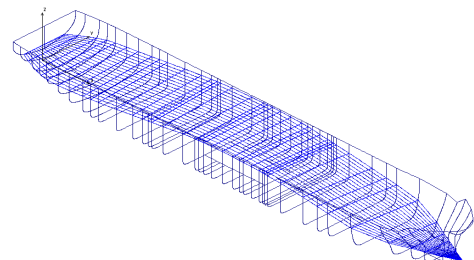
**Fig.7.a.6**  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=75deg.$ , hogging



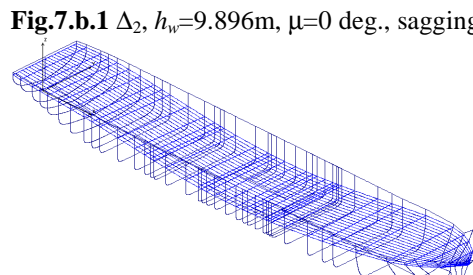
**Fig.7.a.2**  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=15deg.$ , hogging



**Fig.7.b.1**  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=0 deg.$ , sagging



**Fig.7.a.3**  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=30deg.$ , hogging



**Fig.7.b.2**  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=15deg.$ , sagging



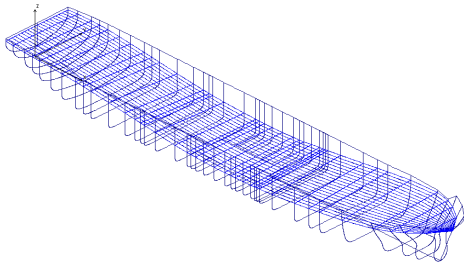


Fig.7.b.3  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=30deg.$ , sagging

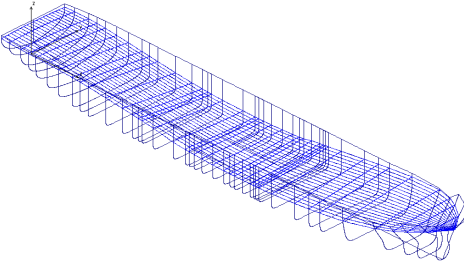


Fig.7.b.4  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=45deg.$ , sagging

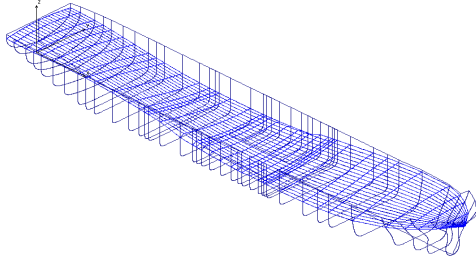


Fig.7.b.5  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=60deg.$ , sagging

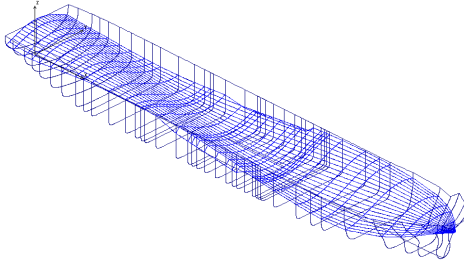


Fig.7.b.6  $\Delta_2$ ,  $h_w=9.896m$ ,  $\mu=75deg.$ , sagging

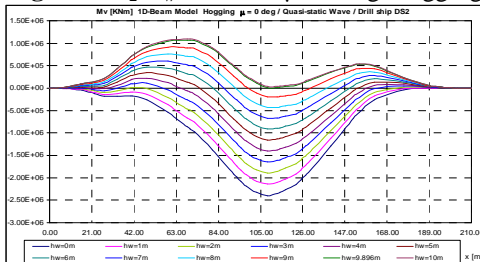


Fig.8.a.1 Case  $\Delta_2, M_v$ [kNm],  $\mu=0 deg.$ , hogg.

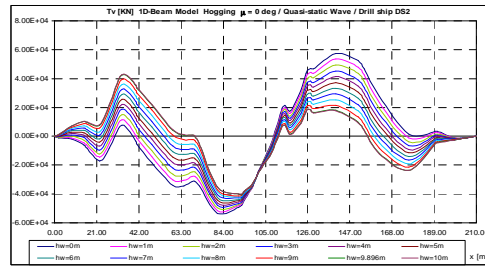


Fig.9.a.1 Case  $\Delta_2, T_v$ [kN],  $\mu=0 deg.$ , hogg.

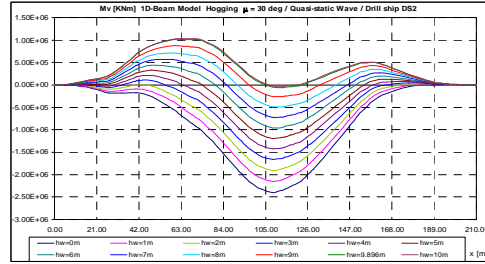


Fig.8.a.2 Case  $\Delta_2, M_v$ [kNm],  $\mu=30 deg.$ , hogg.

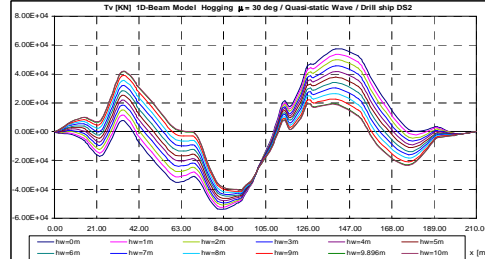


Fig.9.a.2 Case  $\Delta_2, T_v$ [kN],  $\mu=30 deg.$ , hogg.

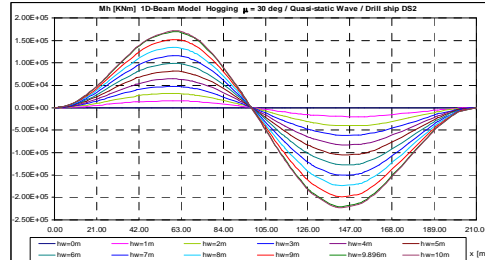


Fig.10.a.2 Case  $\Delta_2, M_h$ [kNm],  $\mu=30 deg.$ , hogg

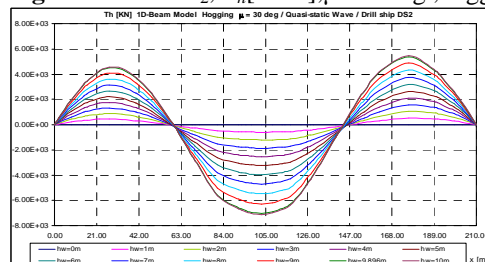


Fig.11.a.2 Case  $\Delta_2, T_h$ [kN],  $\mu=30 deg.$ , hogg

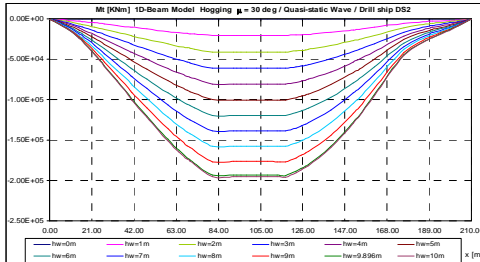


Fig.12.a.2 Case  $\Delta_2, M_i$  [kNm],  $\mu=30$  deg., hogg

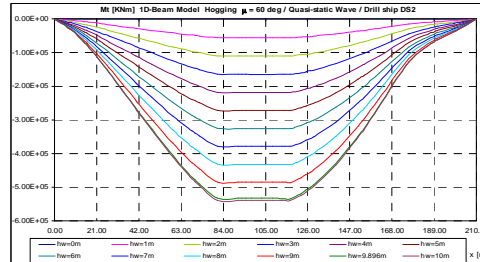


Fig.12.a.3 Case  $\Delta_2, M_i$  [kNm],  $\mu=60$  deg., hogg

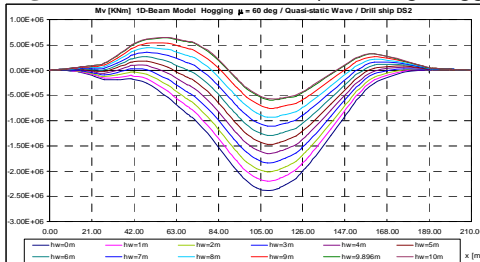


Fig.8.a.3 Case  $\Delta_2, M_v$  [kNm],  $\mu=60$  deg., hogg.

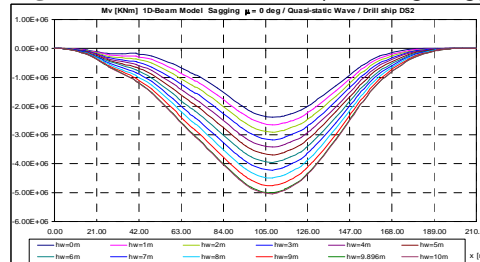


Fig.8.b.1 Case  $\Delta_2, M_v$  [kNm],  $\mu=0$  deg., sagg.

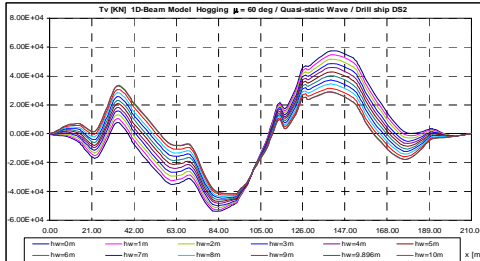


Fig.9.a.3 Case  $\Delta_2, T_v$  [kN],  $\mu=60$  deg., hogg.

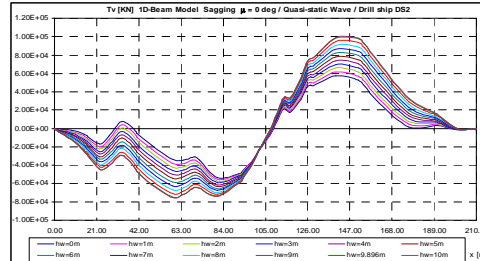


Fig.9.b.1 Case  $\Delta_2, T_v$  [kN],  $\mu=0$  deg., sagg.

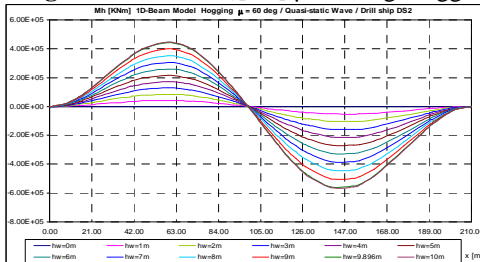


Fig.10.a.3 Case  $\Delta_2, M_i$  [kNm],  $\mu=60$  deg., hogg

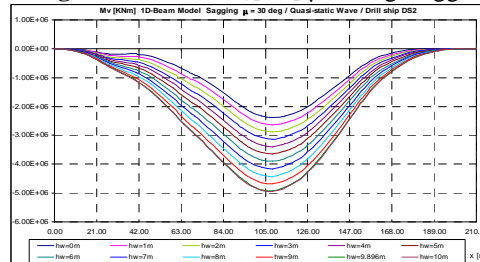


Fig.8.b.2 Case  $\Delta_2, M_v$  [kNm],  $\mu=30$  deg., sagg

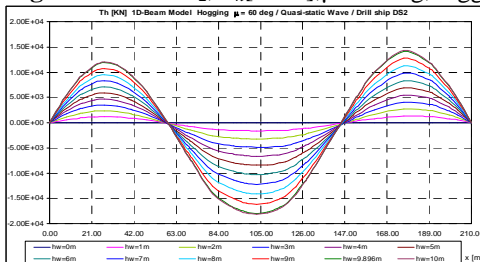


Fig.11.a.3 Case  $\Delta_2, T_i$  [kN],  $\mu=60$  deg., hogg

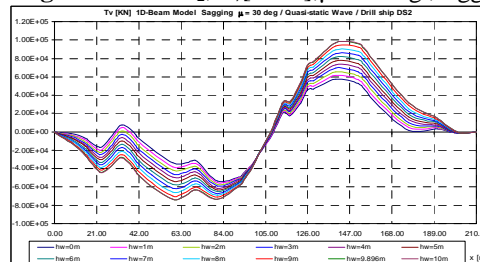


Fig.9.b.2 Case  $\Delta_2, T_v$  [kN],  $\mu=30$  deg., sagg.



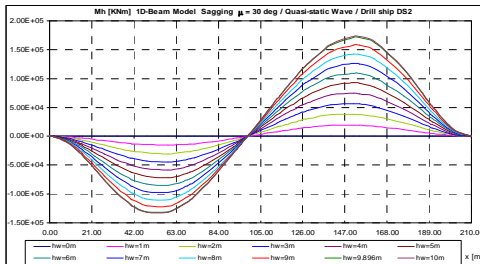


Fig.10.b.2 Case  $\Delta_2, M_h$  [kNm],  $\mu=30$  deg., sagg

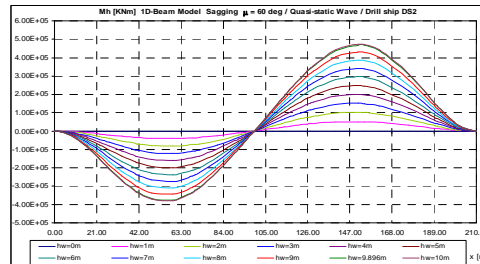


Fig.10.b.3 Case  $\Delta_2, M_h$  [kNm],  $\mu=60$  deg., sagg

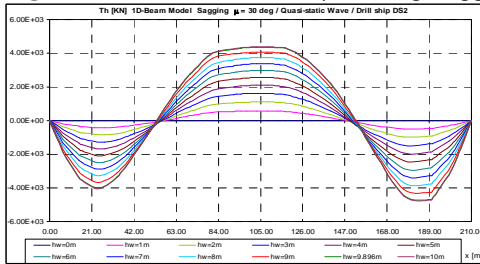


Fig.11.b.2 Case  $\Delta_2, T_h$  [kN],  $\mu=30$  deg., sagg.

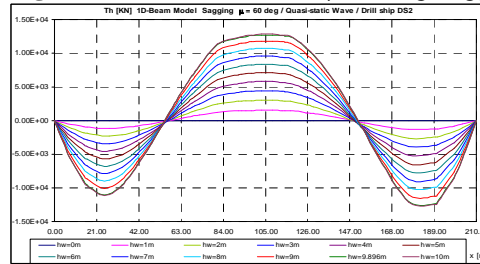


Fig.11.b.3 Case  $\Delta_2, T_h$  [kN],  $\mu=60$  deg., sagg.

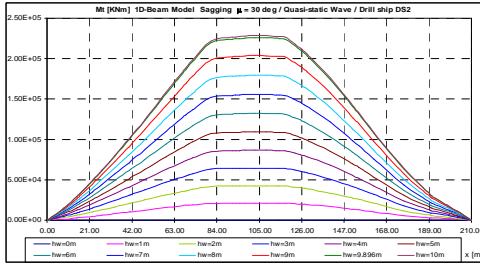


Fig.12.b.2 Case  $\Delta_2, M_t$  [kNm],  $\mu=30$  deg., sagg

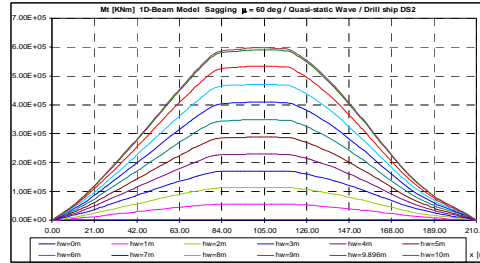


Fig.12.b.3 Case  $\Delta_2, M_t$  [kNm],  $\mu=60$  deg., sagg

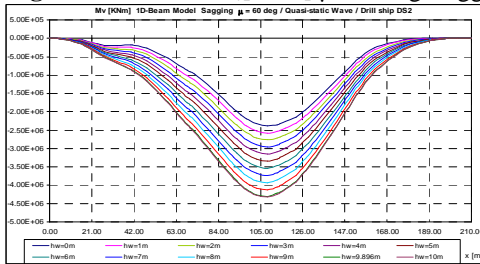


Fig.8.b.3 Case  $\Delta_2, M_v$  [kNm],  $\mu=60$  deg., sagg.

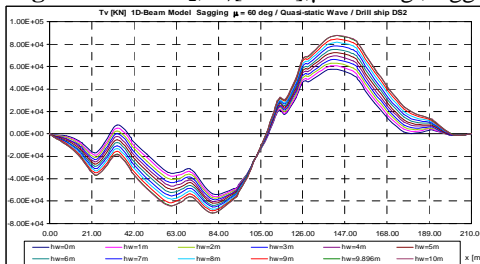


Fig.9.b.3 Case  $\Delta_2, T_v$  [kN],  $\mu=60$  deg., sagg.

## 5. CONCLUSIONS

From Table 3 and Table 5, Figs.2-6 and Figs.7-12, the maximum values for sectional efforts are selected in Tables 6,7, including the rules admissible values [1],[5],[9],[11].

Table 6. Maximum and admissible values,  $\Delta_1$

SW+EDW	max	BV	DNV	GL	ABS
$M_v$ [kNm]	3.73E+64	1.3E+64	4.03E+64	8.6E+64	4.03E+06
$T_v$ [kN]	8.01E+47	2.26E+46	1.8E+46	8.36E+46	1.8E+04
$M_h$ [kNm]	5.56E+51	1.13E+61	1.17E+61	2.06E+61	1.43E+06
$T_h$ [kN]	1.82E+41	1.61E+41	1.74E+41	3.07E+41	4.13E+04
$M_t$ [kNm]	6.84E+53	3.53E+53	3.78E+53	5.27E+53	5.35E+05
SW	max	BV	DNV	GL	ABS
$M_{sw}$ [kNm]	1.20E+61	1.75E+61	1.65E+61	2.48E+61	1.65E+6
$T_{sw}$ [kN]	3.96E+44	4.17E+43	3.09E+43	5.27E+43	3.09E+4

**Table 7.** Maximum and admissible values,  $\Delta_2$ 

SW+EDW	max	BV	DNV	GL	ABS
$M_v$ [kNm]	5.01E+6	4.17E+6	4.04E+6	4.88E+6	4.04E+6
$T_v$ [kN]	9.83E+4	7.34E+4	6.22E+4	8.38E+4	6.22E+4
$M_h$ [kNm]	6.24E+5	1.30E+6	1.26E+6	2.19E+6	1.64E+6
$T_h$ [kN]	2.04E+4	1.84E+4	1.88E+4	3.27E+4	2.44E+4
$M_t$ [kNm]	6.89E+5	3.77E+5	4.01E+5	5.22E+5	5.29E+5
SW	max	BV	DNV	GL	ABS
$M_{sw}$ [kNm]	2.39E+6	1.78E+6	1.65E+6	2.48E+6	1.65E+6
$T_{sw}$ [kN]	5.75E+4	4.23E+4	3.11E+4	5.27E+4	3.11E+4

From the numerical results of this study (Tables 6,7), on global strength of the drillship in oblique EDW waves, the following conclusions are obtained:

- In the case of head and follow EDW waves ( $\mu=0$  deg., hogging and sagging), the vertical bending moments and shear forces maximum values are obtained, on both displacements (Tables 3,5).

- In the case of oblique EDW waves ( $\mu=75$  deg., hogging and sagging), the horizontal bending moments, horizontal shear forces and torsion moments maximum values are obtained, on both displacements (Tables 3,5).

- The maximum still water and EDW wave loads, by global strength analysis, have values around of the shipbuilding rules admissible limits (Tables 6,7).

- Using the equilibrium drillship-EDW wave parameters from this study (Tables 2,4), can be used for 3D-FEM global strength analysis, for stain and stress state assessment.

### Acknowledgements

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