THE ANNALS OF "DUNAREA DE JOS" UNIVERSITY OF GALATI FASCICLE XI – SHIPBUILDING. ISSN 1221-4620, e-ISSN 2668-3156 2020

A NEW ICEBREAKER FOR INLAND WATER CONCEPT

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ABSTRACT

Considering last year's climate changes, we can expect low temperatures during winter, which can lead to the freezing of Danube river and more. This would prevent the freight traffic in Danube's ports meaning that the riparian economy would suffer. This paper presents a new concept of breaking the ice. Using an existent tug, we propose the attachment of an icebreaking additional structure. The paper further describes the concept and includes the necessary calculation methods used for the technological design of this ensemble. This concept presents numerous advantages, including the financial aspect.

Keywords: ice breaker, concept, structural analysis.

1. INTRODUCTION

Currently, the ice breaking operation on inland waters is realised with specialised ships. These icebreakers are active only in the winter season, spending the rest of the year at the quay. Considering the necessity of a permanent crew on board (the abandon of the ship role has to be respected as specified in the international conventions), the costs of energy and other necessities are not justified for the entire year.

Starting from this fact, the paper is proposing a concept of an additional structure, which connected to an inland tug, transforms it into a ship capable of breaking the ice on inland waters.

The concept is based on a tug vessel of enough power reserve to operate in ice infested waters and/or in localised drift ice.

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This removable icebreaking bow can be with or without propulsion. This study analyses the behaviour of an additional structure without propulsion.

This paper contains the shape of the structure, the attachment method to the tug's hull, the modification of the fore part of the tug's hull for strengthening the area of connection and also the analysis of stresses and strains that appear on the tug's hull and on the additional structure's hull.

2. CONCEPT

Considering that the additional structure constitutes a casing for the tug's bow, the designed shape has to respect that desideratum. The aft part of the removable bow will have a shape similar to the shape of the tug's bow.

Fascicle XI

From the beginning, a distance of 500 mm was considered between the two structures as a safety measure, considering a series of factors:

- the additional structure cannot touch the tug's hull during its jigging motion;

- the connection between these structures is accomplished through a rigid massive element;

- localised drift ice cannot enter the empty space between the hulls since it would block the jigging motion of the removable bow.

The connection of the two structures is made with two massive cast steel elements that will be mechanically processed: a throat bolt mounted in the tug's structure and an element in which the bolt is fixed on the exterior of the additional structure's hull.

The concept of the tug – additional icebreaking structure system is presented in figure 1. As the figure illustrates, the hulls are com-

As the figure industrates, the fulls are completely independent and they are connected to each other only if necessary, so that the tug can fulfil its operating functions in port roads.



Figure 1

3. THE REMOVABLE ICEBREAKING BOW

3.1 Shape

The design of the removable bow's shape has to respect the shape of the tug's fore part. It was preferred to adapt the shape of the removable bow to resemble the shape of the tug's bow thus the ensemble's resistance will not increase too much. The body plan obtained after designing the shape of the removable icebreaking bow is given in figure 2.





Figure 2

3.2 Hydrostatics

The body plan is further used for the hydrostatic curves' calculus. The hydrostatic curves diagram is presented in figure 3.



The principal dimensions of the ensemble tug – additional structure (struct) are given in table 1.

One of the main issues of this calculus is determining the ship's resistance.

Table 1										
	Tug	Struct	Complex							
Name	(m)									
Loa	28.1	13.9	35.5							
Lwl	27.8	13.7	34.6							
D	5.35	9.4	7.4							
В	12.6	14	14							
Т	4.3	6.2	4.3							
s	550	550	550							
x _B	14.1	27.2	18.2							
Δ	825	375	1200							

Fascicle XI **3.3 Ship's resistance**

In this case, the ship's resistance was determined with the Holtrop-Mennen method, by transposing the ensemble's dimen-

Та	ble	e 2
1 14	014	

v [kn]	Thrust [kN]	Rt [kN]	PE [kW]	W	t	ЕТАН	ETAR			
9.7	81.82	72.07	359.63	0.12	0.12	1	1.01			

ter.

en in table 2.

3.4 Structural arrangement

To achieve its purpose, the removable bow structure must comply with the classification society rules for icebreaking ships. Therefore, the sizing of the structural arrangement was realised using DNV-GL rules for classification of ships. Thus, the ship's fore part was divided in areas reflecting the magnitude of the ice loads. The hull area divisions are presented in figure 4.





Following the sizing calculus, a midship section of the additional structure was provided in figure 5a. An isometric view of the designed structural arrangement is given in figure 5b.





The sizing calculus is further used to determine the additional structure's total mass. The mass value obtained for the

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the Naval Hydrodynamics Research Cen-

The characteristic values obtained are giv-

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removable bow structure is 177,9 t. Considering the tug's mass value is 298 t, the additional structure's mass represents 67.51% of the tug's mass.

For the connection of the two structures a bolt – connection element system is designed to be mounted on the tug's structure respectively on the removable bow's structure. The designed connection system is presented in figure 6.



4. ANALYSIS OF STRESSES AND STRAINS THAT APPEAR IN THE TUG'S BOW STRUCTURE

As it was stated in the introduction, the ensemble is made of an existent tug (Damen 2910 type) to which is attached the additional icebreaking structure. In order to verify if by attaching the removable bow, the tug's fore structure resists, a stress analysis is necessary. After strengthening the bolt mounting area, the meshing of the tug's structure was realised for the analysis of stresses and strains with the finite element analysis software, FEMAP.

The areas strengthened for mounting the bolt are presented in figure 7. The tug's strengthened area is given in figure 7a and the additional structure's strengthened area in figure 7b.



a)





The mesh area is presented in figure 7a. The mesh was realised using shell elements with 8 degrees of freedom.

The tug's bow structure mesh is given in figure 8. The structure's connections can be observed in figure 8c. The contour of the structure was considered constrained, thus resembling reality as the strengthening of the stiffened area.







The calculus results obtained are presented in figure 9. As shown, the maximum stress value of around 130 MPa is well below the allowable value of the material used for the tug's structure. This area being under stressed means that it is sufficiently well stiffened. The total displacements are also found at low values: 0,6 mm.



a) Von Mises Stress



b) Total Translation Figure 9

5. CONCLUSIONS

The inland icebreaking ensemble proposed in this paper, provides economic advantages, and meets the resistance criteria required for this type of ships.

The stress and strain values obtained from the numerical analysis are within the allowable limits for such floating structures. This concept can be easily configured so

that the operation is simple and qualified staff is not necessary on board.

This concept can be implemented by the companies owning inland tugboats.

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Paper received on August 31th, 2020